

Rutgers Physics 382 Mechanics II (Spring'16/Gershtein)

Class Exam - March 24, 2016

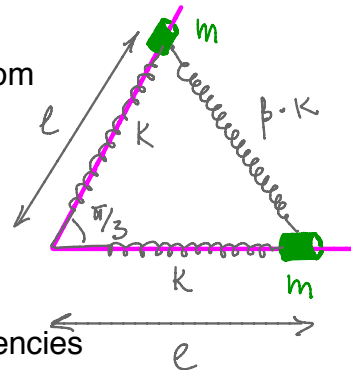
This is a closed book/notes exam. A one-sided 8.5x11 sheet with only formulae is allowed. Please attach the sheet to your solutions. Calculators are not needed. Exam duration - 1 hr 20min. Solve all problems (30 points total)

1. Two masses m can slide without friction on two rods, connected as shown in the figure below, with opening angle $\pi/3$ between them. The masses are attached to the point where the rods are joined with springs (both of length l and coefficient k). The masses are also connected to each other with the third spring, of the same length l and different constant, $k_1 = \beta \cdot k$

a) (3 pts) prove that for small deviations of the masses from equilibrium positions ($x, y \ll l$) the distance between the masses $a = l + (x+y)/2$

b) (5 pts) write down Hamiltonian equations for the system, assuming small deviations from equilibrium

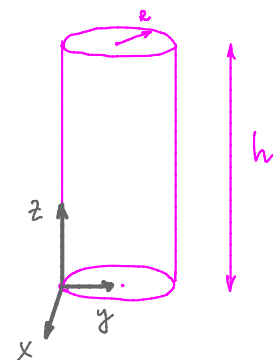
c) (7 pts) Find and describe the normal modes and frequencies



2. (5 pts) Differential cross section of a scattering process is given by $\frac{d\sigma}{d\Omega} = A^2 \cdot (\sin \varphi)^2$. Calculate the total cross section.

3. (5 pts) A thin tube has radius R , mass M , and height h . Origin is on the rim, with z axis parallel to the cylinder and y axis pointing towards the cylinder axis (see figure)

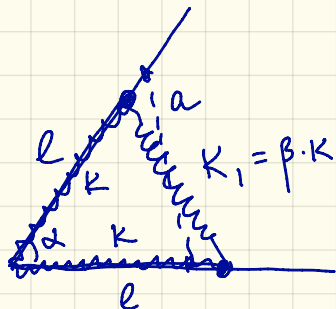
Find all non-diagonal elements of the inertia tensor (i.e. products of inertia)



4. (5 pts) Euler equations with zero torque can be written as $\dot{\vec{L}} + \vec{\omega} \times \vec{L} = 0$. Prove that the magnitude of L is constant.

(1)

a)



$$a^2 = (l+x)^2 + (l+y)^2 - 2(l+x)(l+y) \cdot \frac{1}{2} =$$

$$= l^2 + 2lx + l^2 + 2ly - l^2 - lx - ly =$$

$$= l^2 + l(x+y)$$

(neglect x^2, y^2 terms)

$$a = l \sqrt{1 + \frac{x+y}{l}} = l \left(1 + \frac{x+y}{2l} \right) = l + \frac{x+y}{2}$$

$$b) \quad U = \frac{1}{2} K x^2 + \frac{1}{2} K y^2 + \frac{1}{2} K_1 \left(\frac{x+y}{2} \right)^2$$

$$\mathcal{H} = \frac{p_x^2}{2m} + \frac{p_y^2}{2m} + \frac{1}{2} K x^2 + \frac{1}{2} K y^2 + \frac{1}{8} K_1 (x+y)^2$$

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$$\frac{\partial \mathcal{H}}{\partial p_x} = \frac{p_x}{m} = \dot{x}$$

$$\frac{\partial \mathcal{H}}{\partial p_y} = \frac{p_y}{m} = \dot{y}$$

$$\frac{\partial \mathcal{H}}{\partial x} = Kx + \frac{1}{4} \beta K (x+y) = xK \left(1 + \frac{1}{4} \beta\right) + \frac{1}{4} \beta K \cdot y = -\dot{p}_x = -m\ddot{x}$$

$$\frac{\partial \mathcal{H}}{\partial y} = Ky + \frac{1}{4} \beta K (x+y) = \frac{1}{4} \beta K \cdot x + Ky \left(1 + \frac{1}{4} \beta\right) = -\dot{p}_y = -m\ddot{y}$$

$$\begin{cases} m\ddot{x} = -Kx \left(1 + \frac{\beta}{4}\right) - Ky \cdot \frac{\beta}{4} \\ m\ddot{y} = -Kx \cdot \frac{\beta}{4} - Ky \left(1 + \frac{\beta}{4}\right) \end{cases}$$

$$\begin{pmatrix} 1 & 0 \\ 0 & 1 \end{pmatrix} \ddot{Q} = -\omega_0^2 \begin{pmatrix} 1 + \beta/4 & \beta/4 \\ \beta/4 & 1 + \beta/4 \end{pmatrix}, \quad Q = \begin{pmatrix} x \\ y \end{pmatrix}, \quad \omega_0^2 = \frac{K}{m}$$

$$c) \quad M \ddot{Q} = -KQ \rightarrow \det(K - \omega^2 M) = 0 \quad \omega_0^2 = \frac{\kappa}{m}$$

$$\begin{vmatrix} (1 + \beta/4) \omega_0^2 - \omega^2 & \frac{\beta}{4} \omega_0^2 \\ \frac{\beta}{4} \omega_0^2 & (1 + \frac{\beta}{2}) \omega_0^2 - \omega^2 \end{vmatrix} = 0 = \left[(1 + \frac{\beta}{4}) \omega_0^2 - \omega^2 \right]^2 - \left[\frac{\beta}{4} \omega_0^2 \right]^2 =$$

$$= \left[\left(1 + \frac{\beta}{4} \right) \omega_0^2 - \omega^2 - \frac{\beta}{4} \omega_0^2 \right] \left[\left(1 + \frac{\beta}{4} \right) \omega_0^2 - \omega^2 + \frac{\beta}{4} \omega_0^2 \right] =$$

$$= (\omega_0^2 - \omega^2) \left(\left(1 + \frac{\beta}{2} \right) \omega_0^2 - \omega^2 \right) = 0$$

$$\omega_1 = \omega_0$$

$$\omega_2 = \omega_0 \sqrt{1 + \frac{\beta}{2}}$$

$$\omega_1^2 = \omega_0^2:$$

$$\omega_0^2 \frac{\beta}{4} \begin{pmatrix} 1 & 1 \\ 1 & 1 \end{pmatrix} \begin{pmatrix} x_1 \\ y_1 \end{pmatrix} = 0 \Rightarrow x_1 = -y_1$$

→ opposite phases, the third spring is irrelevant

$$\omega_2^2 = \omega_0^2 \left(1 + \frac{\beta}{2}\right)$$

$$\omega_0^2 \frac{\beta}{4} \begin{pmatrix} -1 & 1 \\ 1 & -1 \end{pmatrix} \begin{pmatrix} x_2 \\ y_2 \end{pmatrix} = 0 \Rightarrow x_2 = y_2$$

→ same phases.

②

$$d\sigma = A^2 \sin^2 \varphi \, d\Omega$$

$$d\Omega = \sin \theta \, d\theta \, d\varphi$$

$$\sin^2 \varphi = \frac{1}{2} - \cos 2\varphi$$

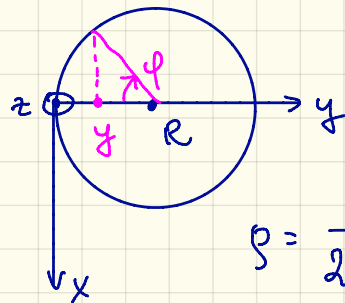
$$\sigma = A^2 \int_0^{2\pi} \int_0^{2\pi} \sin \theta \, d\theta \, \sin^2 \varphi \, d\varphi =$$

$$= A^2 \cdot 2 \cdot \int_0^{2\pi} \sin^2 \varphi \, d\varphi = A^2 \cdot 2 \cdot 2\pi \cdot \frac{1}{2} = 2\pi A^2$$

(3)

$$I_{xy} = 0 \leftarrow \text{symmetry}$$

$$I_{xz} = 0 \leftarrow \text{symmetry}$$



$$\rho = \frac{m}{2\pi R \cdot h}$$

$$y = R(1 - \cos \varphi)$$

$$dm = \rho \cdot R d\varphi \cdot dz$$

$$I_{yz} = - \int dm \cdot y \cdot z = - \int_0^h \int_0^{2\pi} \rho R (1 - \cos \varphi) z \cdot R d\varphi dz$$

$$= - \rho R^2 \frac{h^2}{2} \int_0^{2\pi} (1 - \cos \varphi) d\varphi = \boxed{-\frac{1}{2} m R h = I_{yz}}$$

Another way to solve (suggested by one of the students)
 at the cylinder center, all non-diagonals are zero
 Generalized parallel axis theorem (problem 10.24)

$$I_{yz} = I_{yz}^{cm} - m \cdot \frac{h}{2} \cdot R = \underline{\underline{-\frac{1}{2} m R h}}, \text{ same as above.}$$

$$(4) \quad \dot{\vec{L}} + \vec{\omega} \times \vec{L} = 0$$

multiply by \vec{L}

$$\vec{L} \cdot \dot{\vec{L}} + \vec{L} \cdot (\vec{\omega} \times \vec{L}) = 0$$

$\vec{L} = 0$, since $\vec{\omega} \times \vec{L}$ is $\perp \vec{L}$

$$\vec{L} \cdot \dot{\vec{L}} = \frac{1}{2} \frac{d}{dt} (\vec{L} \cdot \vec{L}) = 0$$

$$\vec{L} \cdot \vec{L} = \text{const.}$$

$$|\vec{L}| = \text{const.}$$